Research Article

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Influence of the blade size on the dynamic characteristic damage identification of wind turbine blades

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Abstract: In this article, the sensitivity coefficients of dynamic characteristic damage identification of blades with different sizes were investigated. The results show that the first third-order vibration modes of the blade before and after damage are consistent, and the wind turbine blade size has no significant effect on the vibration mode; with the increase of the blade size, the first-, third- and fourth-order natural frequency sensitivity coefficients decrease gradually, while the second-, fifthand sixth-order natural frequency sensitivity coefficients increase gradually; as the blade size increases, the thirdorder displacement mode sensitivity coefficient of the blade gradually increases, which indicates that the displacement modes identification effect is better with the increase of the blade size. With the increase of the blade size, the first- and third-order strain modal sensitivity coefficients increase gradually, which indicates that when using first- and third-order strain modes for damage identification, the larger the blade size, the better the identification effect; with the increase of the blade size, the second-order strain modal sensitivity coefficient decreases gradually, which indicates that when using second-order strain mode for damage identification, the larger the blade size, the worse the identification effect. This research could provide a theoretical basis for the application of the dynamic characteristic damage identification method in the damage identification of wind turbine blades of different sizes.

Keywords: wind turbine blade, different blade sizes, blade damage, dynamic characteristics, damage identification

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1 Introduction

Sufficient energy is an important guarantee for modern social progress and economic development [1]. The longterm use of petrochemical energy as the main body of the world's energy has brought a series of problems, such as the exhaustion of petrochemical energy, environmental pollution, ozone layer destruction, the greenhouse effect, etc., and the energy problem has become a major threat to human survival and development. Therefore, renewable energy sources such as solar energy, wind energy, ocean energy, and geothermal energy are gradually being developed and utilized [2]. Wind energy is richer and cleaner than other renewable energy sources, leading to increasing research to make it more efficient [3]. Wind turbines can convert wind energy into electricity, which is the main way to utilize wind energy [4]. Wind turbines have attracted increasing attention due to their advantages of the short infrastructure construction period, flexibly installed capacity, and no environmental pollution

In recent years, with the increasing depletion of fossil fuels and the increasingly prominent environmental problems, investment in wind energy has begun to increase, and the wind power industry has developed rapidly and occupies an important position in renewable energy [6]. Since 2001, the global wind power industry has developed rapidly, and the annual growth rate of the global wind power installed capacity has reached 20–30%. By the end of 2020, the cumulative installed capacity of global wind power reached 733.28 GW.

Blades are the main components of wind turbines, which play an important role in capturing wind energy [7]. Whether the blades can work normally has a very important impact on the entire wind turbine [8]. The working environment of wind turbine blades is harsh, and small debris such as sand and gravel mixed in the wind, as well as wind and snow, lightning, *etc.*, will cause damage to the blades to a varying degree [9].

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In addition, the blade is subject to the combined action of alternating load and random load for a long time when it is in a working state, which makes it prone to fatigue damage [10]. Once the blade is damaged in operation, the damage will continue to develop under the action of alternating loads, and when the damage develops to a certain extent, the blade will be broken [11]. This not only affects the normal operation of the wind turbine but also causes damage to the entire unit, which will bring economic losses to the wind farm, and seriously affects the application and promotion of wind power generation.

If the initial blade damage can be timely detected and measures are taken for maintenance, a series of losses caused by the blade breaking can be avoided [12]. However, it is hard to find blade damage in the early stage. The blade is installed at the top of the wind turbine tower, which brings a lot of inconvenience to the detection work. Therefore, effective wind turbine blade damage detection has become a problem worthy of further study.

Wind turbine blade damage identification is to study the response of blade structure under external excitation. Through structural response analysis, various characteristic parameters of the blade structure are obtained and compared with the design data to determine the performance and state of the blade structure. Scholars have systematically studied the damage identification of wind turbine blades and achieved many reliable research results. Ghoshal *et al.* [13] applied a piezoelectric ceramic actuator (PZT) sensing system to blade damage identification and motivating blade vibration to detect blade damage through PZT. Sundaresan et al. [14] preset the PZT sensing system inside the wind turbine blade to monitor the damage to the blade in real time. In addition, scholars have studied the blade damage identification of wind turbines based on acoustic emission technology [15-18]. Zhu et al. [19] combined acoustic emission technology with the PZT sensor for damage identification of wind turbine blades. Smith et al. [20] and Beattie and Rumsey [21] identified structural damage according to the temperature change of wind turbine blades. Xiao and Yan [22] applied infrared imaging technology to the nondestructive testing of wind turbine blades.

In recent years, many scholars in China have made some achievements in the detection of blade damage through blade dynamic characteristics [23–25]. Compared with static measurement data, dynamic measurement data were more abundant and studied in-depth [26]. Therefore, damage identification methods based on dynamic properties have developed rapidly in recent years and have been widely used in practical engineering [27].

From the dynamic point of view, the general structural system was regarded to be composed of a mass matrix, stiffness matrix, damping matrix, etc. When the structure was damaged, it usually caused changes in the physical parameters of the structure, and the modal parameters were based on the physical parameters of the structure. When the physical parameters of the structure change, the modal parameters (natural frequency, mode shape, etc.) will also change accordingly. Therefore, the damage to the structure could be identified by the change in the modal parameters (natural frequency, mode shape, etc.). When the damage identification of the structure is carried out, the dynamic characteristic analysis of the undamaged structure should be carried out to obtain its modal parameters and the modal parameters should be used as the benchmark. The modal parameters of the damaged structure were then obtained and compared with the benchmark parameters so that the structural damage could be identified [28].

The finite element method could reasonably simplify the complex problem of the structural system, which could solve practical problems more conveniently [29,30]. At present, converting the actual structure into a finite element model and using it as a reference has been used in most structural damage identification methods.

Structural damage identification mainly included damage identification, damage localization, and damage degree assessment [31–33]. Generally, the simplest method was used to determine whether the structure was damaged or not, and then a more complex but accurate identification method was used to further identify the structure and then the location of the damage and the degree of damage were judged [34–36].

To meet the different needs of society, the power of wind turbines was varied and the blade size was also different. The change in the blade size may have a certain impact on the damage identification effect. At present, several studies have been carried out on the damage detection of wind turbine blades. However, most of the existing studies focus on single-size blades, without considering the effect of blade size on damage identification [37]. In this article, different sizes of blades were selected for modeling, and the finite element analysis software ANSYS was used to analyze the dynamic characteristics of blades in shutdown and running states. The sensitivity coefficients of the dynamic characteristics damage identification of blades with different sizes were investigated, and the results were compared to explore the influence of different blade sizes on the damage identification effect of dynamic characteristics.

2 The wind turbine blade model

2.1 Blade size and material properties

According to the data provided by a wind power equipment company, the blade sizes corresponding to different power wind turbines are shown in Table 1. To study the influence of different blade sizes on the damage identification results of dynamic characteristics, wind turbine blades with lengths of 2.7, 3.2, 3.7, 4.5, and 6.5 m were selected for modeling and analysis. The material properties of wind turbine blades according to Su *et al.* [37] are shown in Table 2.

2.2 The finite element model of the blade

The finite element model of the blade is shown in Figure 1. The length of the blade was 4.5 m and the widths were 22.5 cm at the tip and 71.0 cm at the widest part of the

Table 1: Blade size of wind turbines with different powers

| Wind turbine | 1 | 3 | 5 | 10 | 15 | 20 | 30 | 40 |
|------------------------------|-----|-----|-----|-----|-----|-----|-----|------|
| power (kW) Blade size (m) | 1.3 | 1.9 | 2.7 | 3.2 | 3.7 | 4.5 | 6.5 | 12.6 |

blade. The blade damage unit in the 4.5 m blade is shown in Figure 2 (the damage unit size is $12.2 \, \text{cm} \times 24.3 \, \text{cm}$). The damage units at the same position are selected for the analysis of other size blades, and the size of the damage units increased (decreased) in proportion to the increase (decrease) of the blade size. In this article, the initial mesh length of the blade was 40 mm; this was adjusted according to the accuracy requirements of the results and the size of the blade.

The damage degrees of the blade are 0, 20, 40, 60, and 80%. The different damage degrees of the blade are simulated by adjusting the elastic modulus of the blade (E is the elastic modulus without damage and E' is the elastic modulus with damage) [37], as follows:

$$\lambda = \frac{E - E'}{E}.\tag{1}$$

3 Recognition of the blade damage

3.1 Mode shape of the blade

The modal analysis of wind turbine blades with different sizes before and after the damage was carried out by ANSYS. The first third-order vibration modes of the blade before and after the damage were consistent: the firstorder vibration mode was flapping vibration; the second-

Table 2: Material properties of wind turbine blades

| Material type | Span modulus (GPa) | Radial modulus (GPa) | Shear modulus (GPa) | Poisson's ratio |
|--------------------------------|--------------------|----------------------|---------------------|-----------------|
| Glass fiber-reinforced plastic | 42.6 | 16.5 | 5.5 | 0.22 |



Figure 1: Finite element model of the blade.



Figure 2: Damaged area of the blade.

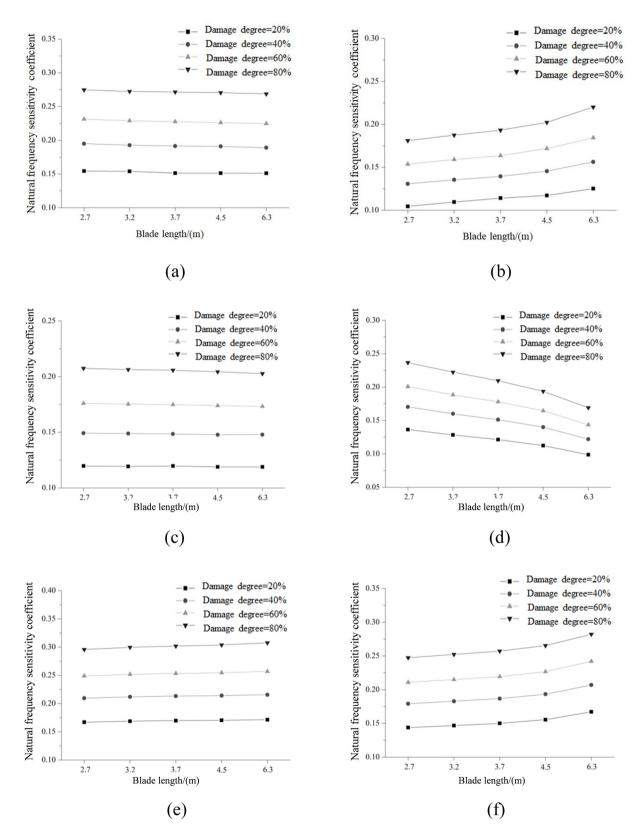


Figure 3: Sensitivity coefficient of natural frequency under the shutdown state: (a) first order, (b) second order, (c) third order, (d) fourth order, (e) fifth order, and (f) sixth order.

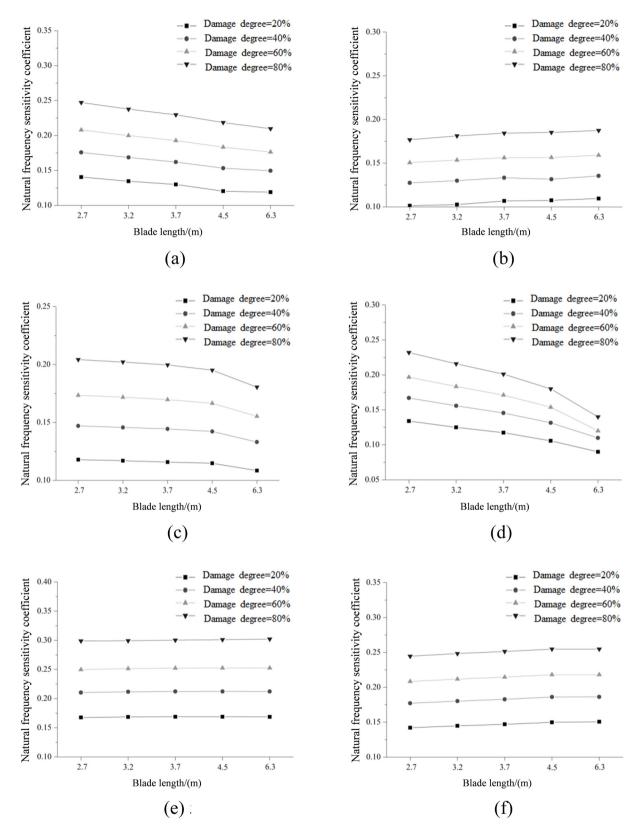


Figure 4: Sensitivity coefficient of natural frequency under the running state: (a) first order, (b) second order, (c) third order, (d) fourth order, (e) fifth order, and (f) sixth order.

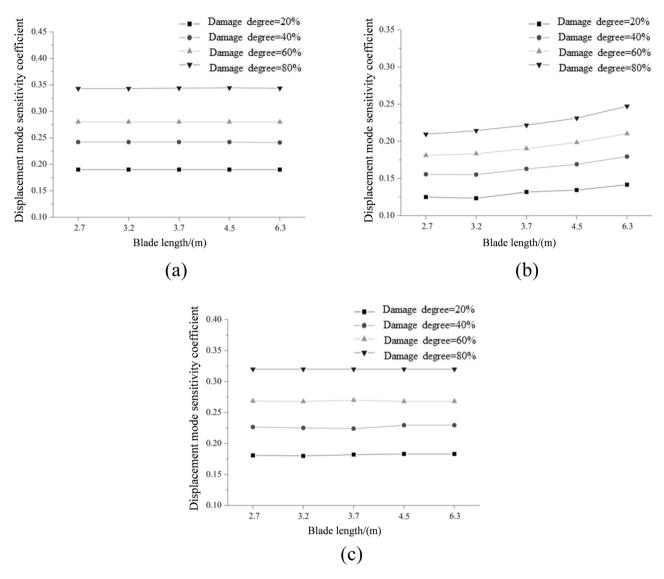


Figure 5: Sensitivity coefficient of the displacement mode under the shutdown state: (a) first order, (b) second order, and (c) third order.

order vibration mode was vibration; and the third-order vibration mode was flapping vibration. The size of the wind turbine blade has no significant effect on the vibration mode.

3.2 Sensitivity analysis of damage identification based on the natural frequency

The natural frequency sensitivity coefficient of the blade is as follows (f_i is the natural frequency without damage and f'_i is the natural frequency with damage) [37]:

$$Q_f = \sqrt[4]{\frac{f_i - f_i'}{f_i}}. (2)$$

Figures 3 and 4 show the sixth-order natural frequency sensitivity coefficients of blades with different sizes under shutdown and running states, respectively. It can be seen that the sixth-order natural frequency sensitivity coefficient of the blade with different sizes under shutdown and running states (the selected speed rate: 17 rad/s [37]) increased with the increase of the blade damage. With the increase of blade size, the first-, third-, and fourth-order natural frequency sensitivity coefficients decreased gradually, while the second-, fifth-, and sixth-order natural frequency sensitivity coefficients increased

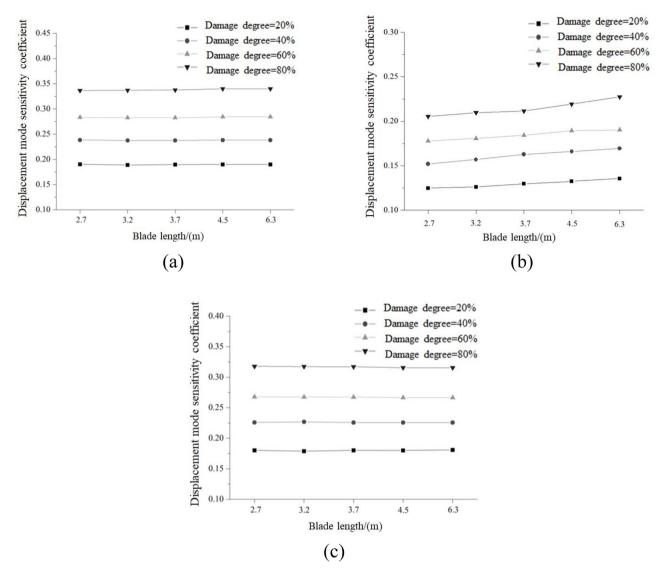


Figure 6: Sensitivity coefficient of the displacement mode under the running state: (a) first order, (b) second order, and (c) third order.

gradually. For damage identification based on natural frequency, first-, third-, and fourth-order natural frequencies can be selected when the blade size is small, while second-, fifth-, and sixth-order natural frequency frequencies can be selected when the blade size is large. When the blade size increased from 2.7 to 6.3 m, the maximum change rate of the sixth-order natural frequency sensitivity coefficients under the shutdown state was 28.60%, while it was 27.37% under the running state.

3.2.1 Sensitivity analysis of damage identification based on the displacement mode

The displacement mode sensitivity coefficient of the blade is calculated as follows ($\varphi_{\rm d}$ is the displacement mode

without damage and φ'_d is the displacement mode with damage) [37]:

$$Q_{\rm d} = \sqrt[4]{\frac{\varphi_{\rm d} - \varphi_{\rm d}'}{\varphi_{\rm d}}}.$$
 (3)

Figures 5 and 6 show the third-order displacement mode sensitivity coefficient of blades with different sizes under shutdown and running states, respectively. It can be seen that the third-order displacement mode sensitivity coefficients of the blade with different sizes under shutdown and running states increased with the increase of the blade damage. It can also be seen that as the blade size increased, the third-order displacement mode sensitivity coefficient of the blade with different sizes gradually increased, which indicated that when using displacement

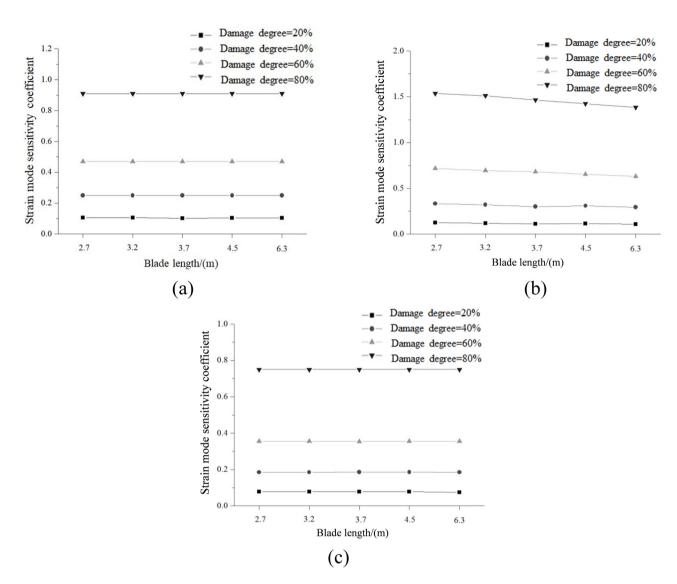


Figure 7: Sensitivity coefficient of the strain mode under a shutdown state: (a) first order, (b) second order, and (c) third order.

modes for damage identification, the identification effect was better with the increase of blade size. When the blade size increased from 2.7 to 6.3 m, the maximum change rate of the third-order displacement mode sensitivity coefficient under the shutdown state was 18.01%, while it was 15.57% under the running state.

3.2.2 Sensitivity analysis of damage identification based on the strain mode

The strain mode sensitivity coefficient of the blade is calculated as follows (φ_s is the strain mode without damage and φ_s' is the strain mode with damage) [37]:

$$Q_{\rm s} = \sqrt[4]{\frac{\varphi_{\rm s} - \varphi_{\rm s}'}{\varphi_{\rm s}}}.$$
 (4)

Figures 7 and 8 show the third-order strain mode sensitivity coefficient of blades with different sizes under shutdown and running states, respectively. It can be seen that the third-order strain mode sensitivity coefficient of the blade with different sizes under shutdown and running states increased with the increase of the blade damage. There was no obvious difference in the strain modal sensitivity coefficient of the blade between the running state and the shutdown state. With the increase of the blade size, the first- and third- order strain modal sensitivity coefficient under the shutdown state and

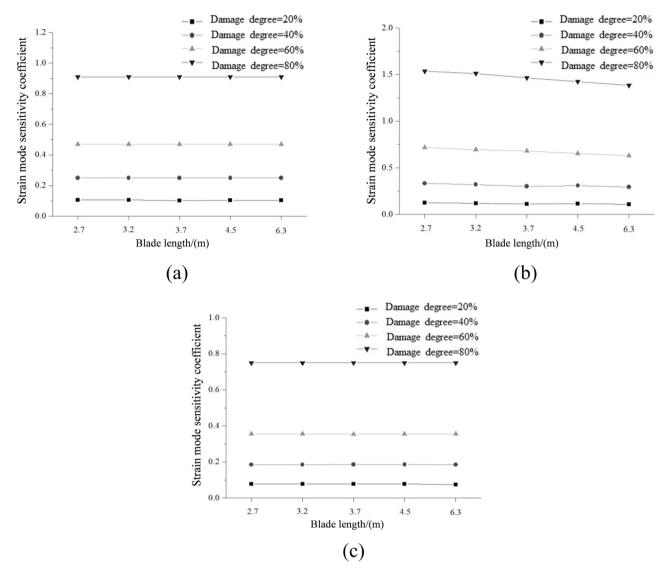


Figure 8: Sensitivity coefficient of the strain mode under a running state: (a) first order, (b) second order, and (c) third order.

running state increased gradually, which indicated that when using first- and third-order strain modes for damage identification, the larger the blade size, the better the identification effect. With the increase of the blade size, the second-order strain modal sensitivity coefficient under the shutdown state and running state decreased gradually, which indicated that when using second-order strain mode for damage identification, the larger the blade size, the worse the identification effect. When the blade size increases from 2.7 to 6.3 m, the maximum change rate of the third-order displacement mode sensitivity coefficient under a shutdown state was 14.26%, while the maximum change rate of the third-order displacement mode sensitivity coefficient under a running state was 15.87%.

4 Conclusions

- The first third-order vibration modes of the blade before and after damage were consistent, and the wind turbine blade size had no significant effect on the vibration mode.
- 2) With the increase of the blade size, the first-, third-, and fourth-order natural frequency sensitivity coefficients decreased gradually, while the second-, fifth-, and sixth-order natural frequency sensitivity coefficients increased gradually.
- 3) With the increase of the blade size, the third-order displacement mode sensitivity coefficient of the blade gradually increased, which indicated that the

- displacement mode identification effect was better with the increase of the blade size.
- 4) With the increase of the blade size, the first- and thirdorder strain modal sensitivity coefficient increased gradually, while the second-order strain modal sensitivity coefficient decreased gradually.

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