

Research Article

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Preparation of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ -based composite PCM and its application in air insulated box

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Abstract: In order to provide continuous supply for succeeding application, abundant electricity energy and solar energy can be stored by means of thermal storage technology. In the present paper, the heat energy storage/exothermic tests are conducted to evaluate the performance of thermal energy storage and release of electricity energy in the self – designed heat storage box equipped with the composite phase change material (PCM), the delivery of heat to surrounding environment is through an air blower directly. The composite PCM consisting of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ and $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ at the ratio of 7:3 is used to carry out the tests, which is with stable solidification property. Following results are obtained from this research: (1) The solidification temperature of the composite PCM is 33.4 °C with reduced supercooling degree of 2.6 °C; (2) The is phase change exothermic enthalpy value of the composite PCM is 178.02 J/g; (3) The self – designed “peak load shifting” heat storage equipment for electricity energy is with the energy exchange efficiency of 89.59%. The achievements of this research show the applicability of the thermal storage technology by means of the composite PCM.

Keywords: energy exchange efficiency; heat energy storage; phase change materials; solidification temperature; supercooling.

Introduction

At present, the excessive use of fossil energy has caused deterioration of environment gradually and shortage of

fossil energy, which in turn become a limiting factor for economic growth. Therefore, the structure of energy resource is facing adjustment worldwide, sustainability and coordination with the environment and resources will be the unavoidable direction in future. The structure change of energy resource has become an important and hot research topic, which involves a number of very complex factors (Hou, Xu, and Cheng 2008). It is necessary to adjust the energy structure so as to demand of the continuous development of economy and environment for human life.

Exploration and application of renewable energy, such as wind energy, solar energy, geothermal energy, et al., has been developed rapidly. Ha et al. proposed a simple and effective trailing edge flap system consisting of a motor-driven worm gear drive and flexible torsion bar (Ha 2020). A preliminary level design study was performed to show the applicability of the new trailing edge flap system for wind turbine rotor blade or helicopter blade.

As alternatively important measurement for sustainable energy development, heat storage technology has received extensive attention. The abundant electricity energy and solar energy can be stored by means of thermal storage technology for its continuous supply in form of heat energy. As an example, the abundant electric energy can be restored in the form of heat in the valley period of electricity consumption, and the restored heat energy can be released during the peak period of electricity consumption, which can be delivered to the user to play the role of “shifting load peak”; In addition, abundant solar energy can also be restored in form of heat during daytime period, and the released when needed. Therefore, heat storage technology has broad development prospects for the implementation of energy structure adjustment, energy saving and emission reduction.

Kıvanç et al. made a tank filled with water and marble as a heat store to replace the general wall (Kıvanç and Wall 2019), the developed Trombe wall system could provide 30% of the energy required for heating and cooling of the building. Wahba et al. employed green roofs/walls to adjust the urban heat island effect by simulation (Wahba et al. 2019), which could adjust the outdoor air temperature

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by 10 °C and improve the outdoor thermal comfort by 2 predicted average values (PMV). Qerimi et al. made solar water heaters to replace the traditional water heaters for 38,289 residential households in Pristine of Kosovo (Qerimi et al. 2020), the initial cost of the solar water heater is 60113730 €, the system saves 7274910 € annually and reduces CO₂ emission by 22973400 kg.

Kibaara et al. developed tool by modeling to assess the techno-economic and the environmental impacts in monetary form (Kibaara et al. 2020), which is able to determine the viability of a plant in a given region.

Phase change material (PCM) is another main thermal energy storage medium, which can be used in heat storage technology. According to its chemical composition, PCM can be classified into organic PCM, inorganic PCM, and composite PCM. Inorganic hydrated salt is a typical PCM.

Research status

Research status of phase change materials (PCMs)

Ahmet et al. studied the thermal properties of composite PCMs of fatty acids and their eutectic mixture attempting to use the heat storage materials in the fields of building (Ahmet 2003; Ahmet and Kamil 2001; Gulseren and Ahmet 2003), low – temperature heating and solar energy systems (Ahmet and Kamil 2006; Kadir et al. 2004).

German researchers have found that paraffin wax could be encapsulated as microcapsules with size of 1–20 µm by poly-methylmethacrylate (PMMA) (Gschwander and Henning 2005; Schossig, Gschwander, and Haussmann 2005). The wall thickness of the microcapsules is less than 20 nm, which is easy to be dispersed in water without agglomeration. In the molten state, there is no interaction between the paraffin microcapsules and the liquid carrier, and the microcapsule is expected to be used in low temperature field.

Walsh et al. used hydrated inorganic for night cold storage (Walsh, Murray, and O'Sullivan 2013), and numerical calculations showed that the operating time can be reduced by 67% during peak electricity consumption.

Xu et al. found that the composite PCM made from paraffin/diatomite has good compatibility with cement (Xu and Li 2014), and the composite material has good structural uniformity with excellent energy storage and thermal insulation performances.

Li et al. analyzed the PCM gypsum board of lauric acid – decanoic acid in the field of building by means of the enthalpy method model (Li, Cheng, and He 2013). Their results show that the latent heat utilization rate of

this gypsum board is 38.7%, which could save energy by 27.6% as compared to the ordinary walls with the same structure.

The application status of PCMs in “peak load shifting” heat storage device of electricity energy

The “peak load shifting” of electricity can be implemented to alleviate the grid load by heat storage equipment. The design and application research of “peak load shifting” equipment with PCMs in domestic is in primary stage, and detailed work needs to be done.

Brousseau and Lacroix applied the multi-layer phase change material system in an electric heating system to meet the demand of balancing the electricity load (Brousseau and Lacroix 1996). The performance of the heat storage unit is simulated with corresponding mathematical model. The results showed that the melting within the PCM is inhomogeneous, especially the material at the bottom is not completely molten even though the temperature at the top of the heat storage unit is very high owing to the improper arrangement of the heater and the heat storage unit.

Xu and Li once analyzed the feasibility of applying energy storage electric heating technology in the Guanzhong area of Shaanxi province of China (Xu and Li 2019). Their research shows that heat storage of electric heating needs a comprehensive consideration of mature technology, safety, and initial installation. It is feasible to promote thermal storage electric heaters. Mazzeo et al. comparatively conducted an experiment employing a PCM with an analytical model so as to carry out the evaluation of the thermophysical properties of the PCM sample, the analytical model constitutes a valid algorithm for the evaluating the latent and sensible contribution and the trend of the position of the bi-phase interface in time (Mazzeo and Oliveti 2017; Mazzeo and Oliveti 2018; Mazzeo et al. 2017). Halford et al. addressed a potential peak air conditioning load shifting strategies using encapsulated phase change materials. The PCM was designed to be installed within the ceiling or wall insulation to perform the delay of the peak air conditioning demand time (Halford and Boehm 2007).

In general, the cost of inorganic PCMs is superior to that of organic PCMs, such as paraffin wax, et al.

In recent years, Zheng et al. improved the thermal storage properties and heating – cooling circling behaviors of Na₂HPO₄·12H₂O based PCM by adding Na₂SO₄·10H₂O (Jin et al. 2019; Zheng, Liu, and Jin 2019a; Zheng et al. 2019b), and anti-corrosive property with addition of Na₂SiO₃·9H₂O by compositing material technology. However, the phase transformation temperature of the Na₂HPO₄·12H₂O based

PCM is around 30 °C (Jin et al. 2019; Zheng, Liu, and Jin 2019a; Zheng et al. 2019b), which is still lower for some usage.

The main content of this paper

This paper aims to conduct preparation $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ -based composite PCM with higher phase transformation temperature by other additive, and characterize performance analysis of by step-cooling test, DTA test, etc., and then the heat storage and release performance of the material in a self manufactured insulated air box, in which the delivery of heat with surrounding environment is through an air blower directly. The flowchart of the research in this paper is shown in Figure 1.

Basic property of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ based composite material

Experimental materials and setups

The experimental materials are shown in Table 1, and the setups used in the experiment are shown in Table 2. The test tubes used in the experiment are all plastic test tubes with a diameter of 20 mm and a volume of 50 mL.

Thermal performance of phase change materials

Disodium hydrogen phosphate dodecahydrate ($\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$) is a typical crystalline hydrated salt used for energy storage material. In recent years, it has attracted more

Table 1: Experimental materials.

Name of material	Grade	Manufacturer
$\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$	Industrial grade, 98% purity, 25 kg/bag	Shifang Xuhongda chemical plant
$\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$	Industrial grade, 98% purity	Shifang Xuhongda chemical plant

attention (Hiran et al. 2015; Huang et al. 2013; Lan et al. 2009), but it has a supercooling of 13 °C, which limits its practical application. One aim of this study is to reduce the supercooling of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ by additive chemicals. The additive $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ is with industrial grade with purity of 98%. From the experimental results of the mass ratios of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ to $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ at 5:5, 8:2, 7:3 and 9:1, it found that the ratio of 7:3 obtained lowest degree of supercooling and promising phase transition temperature. So, the composite of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ with additive $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ at 7:3 is used here for further investigation.

Step cooling curve of the composite

The composite of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ with additive $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ at 7:3 is used here for detailed analysis.

40 g of the composite material is placed into the test tube, which is then sealed with waterproof tape. The test tube is placed in a 60 °C constant temperature water bath for heating until the composite salt molten completely and kept there for half an hour. Then, remove the test tube out of the water bath and place it in a cooling water bath at temperature of 20 °C, as shown in Figure 2. The step – cooling curve (solid process) of the composite salt was by

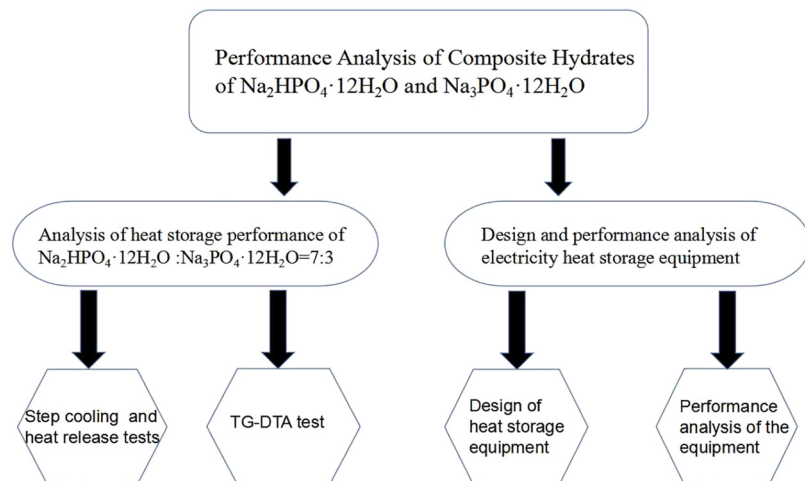


Figure 1: Flowchart of the research in this paper.

Table 2: Experimental setup.

Instrument	Model	Producer	Parameters of performance
Electronic balance	JJ124BC	Changshu Shuangjie Exp. Equip. Factory	Accuracy 0.1 mg, maximum range 200 g
Water bath with heater & stirrer	DF-101S	Gongyi Zhi China Instrument Co., Ltd.	Heating temperature 0–400°
Thermometer	TM-902C	Shanghai Jiamin Instrument Co., Ltd.	Temperature range –50–750 °C
DTA-TG instrument	HCT-1	Beijing Hengjiu Sci. Instru. Factory	
Thermocouple sensor	k	Changzhou Tengde Electronics Co., Ltd.	Range: 50–1200 °C
Single layer glass reactor	DF-10L	Zhengzhou Scitech Biotech. Co., Ltd.	Speed: 1000 g/cm Capacity: 10L

obtained in the cold water bath, and the temperature was recorded every 2 s, which is shown in Figure 3. It can be seen from Figure 3 that the solidification temperature of the composite material is 33.4 °C, the heat release plateau is about 37.5–39 °C with the duration of heat release of 40 min.

DTA –TG test

The DTA test for the composite Na₂HPO₄·12H₂O- additive Na₃PO₄·12H₂O is carried out at the DAT-TG HCT-1 instrument. The temperature of DTA test ranges from 20 to 180 °C at the heating rate of 1 °C/min. The experimental results are shown in Figure 4.

It can be seen from Figure 4 that there are three absorption peaks during the heating process. The peak temperatures are 40.4, 65.2, and 110.8 °C, and the corresponding enthalpy change is 911.44, 342.64 J/g, Na₃PO₄·12H₂O, respectively. There are two weight loss steps for the composite phase change material, i.e., a weight loss step rate of 33.22% ranging 30–65 °C and a weight loss step rate of 22.75% ranging 65–110 °C. Extrapolating the first endothermic peak to the baseline of temperature, it obtains the initial phase transition temperature of the composite phase change material being 36.0 °C, see Figure 4.

As compared with the solidification temperature of the composite phase change material given in Figure 3, i.e., 33.4 °C, the supercooling degree of the composite is 2.6 °C, which indicates that this composite phase change material has lower supercooling degree and promising latent heat performance in the low temperature range. The supercooling degree of this composite phase change material is much smaller than those mentioned in literatures (Hiran et al. 2015; Huang et al. 2013; Lan et al. 2009), so it is suitable for thermal energy storage.

In summary, the Na₂HPO₄·12H₂O based composite material has excellent melting – solidification cycle characteristics, stable phase transition temperature, good heat storage performance, and applicability in low temperature range.

Determination of exothermic quantity of PCM in water during phase change process

The schematic diagram shown in Figure 5 is used to test the exothermic quantity of the composite PCM in water.

The specific steps for determining the exothermic quantity of the composite PCM in water are as follows: place the plastic test tube containing the composite PCM into a constant temperature water bath of 60 °C and stay

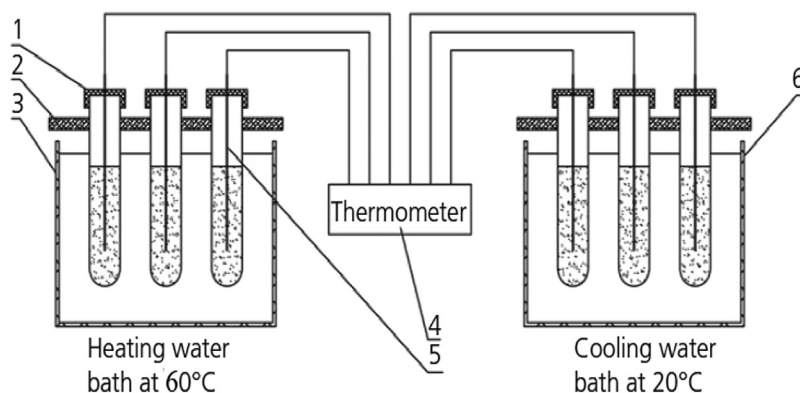


Figure 2: Diagram of melting-solidification test for composite PCM. (1) - tube cover and sealing tape; (2) - test tube clip; (3) - water bath with heater & magnetic stirrer; (4) - thermometer; (5) - thermometer sensor; (6) - cooling water bath.

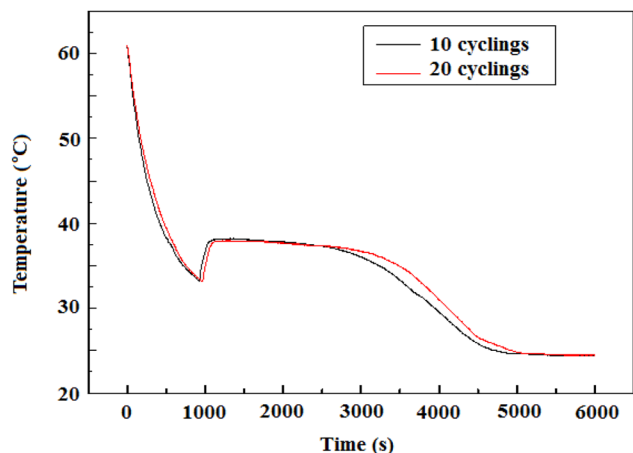


Figure 3: Step cooling curve of the composite $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ - $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$.

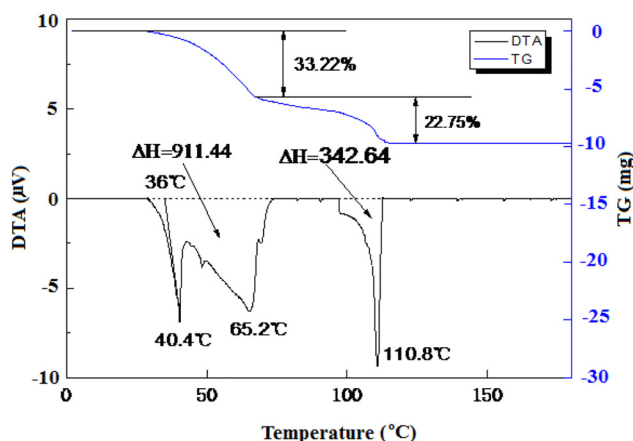


Figure 4: DTA curve of composite $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ -additive $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$.

there inside for 2 h continuously till the composite molten completely first. Then keep the temperature for 30 min; after that remove the test tube quickly from the water bath, and place it into the previously prepared heat preservation containing a certain amount of cold water (16 °C), and record the change of water temperature versus time every 3 s till the temperature of the water going through a peak value.

Equation (1) is the calculation formula for assessing the exothermic quantity of the composite PCM in water,

$$\Delta H = \frac{c_2 \cdot m_2 \cdot (t_2 - t_1) - c_1 \cdot m_1 \cdot (t - t_2)}{m_1} \quad (1)$$

in which c_1 is the specific heat capacity of the composite PCM, m_1 is the mass of the composite PCM, here $m_1 = 20$ g in this test; m_2 is the mass of cold water in this test, here $m_2 = 150$ g; t is the temperature of the constant temperature water bath, here $t = 60$ °C; c_2 is the specific heat capacity of

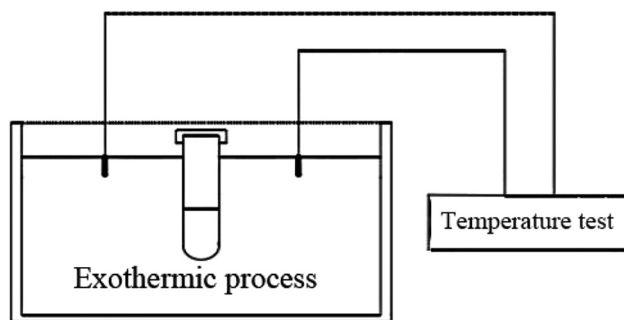


Figure 5: Schematic diagram of exothermic quantity of the composite PCM in water.

cold water; t_1 is the initial temperature of cold water; t_2 is the final temperature of cold water.

From the practical test, the initial temperature of cold water is $t_1 = 16.2$ °C, the final temperature of cold water is $t_2 = 24.1$ °C, the specific heat capacity of water $c_2 = 4.18 \times 10^3$ J/(kg·°C); constant temperature water bath set temperature $t = 60$ °C. Substituting all these data into Eq. (1), the actual the exothermic quantity of the composite PCM in water is obtained, $\Delta H = 178.02$ J/g.

Experimental results for storage electricity energy in heat form and analysis

Heat storage system for electricity energy

The heat storage system for electricity energy in heat form is mainly composed of electric heater, air blower, composite PCM and insulated box, as shown in Figures 6 and 7.

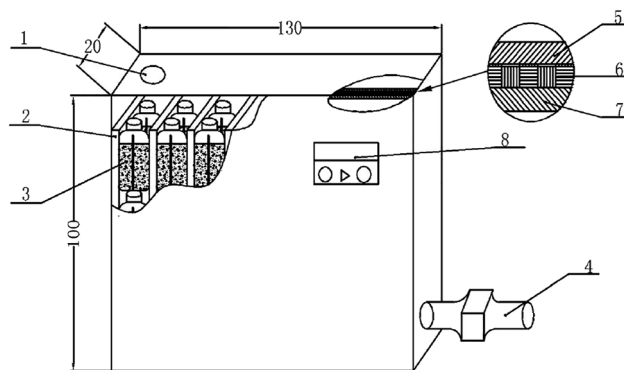


Figure 6: The schematic diagram of the heat storage device for electricity energy. (1) air inlet, (2) electric heater, (3) bottle containing composite PCM, (4) air blower, (5) aluminum silicate fiber, (6) insulation board, (7) Iron sheet, (8) temperature controller.



Figure 7: Photo of the air insulated box for electricity energy.

The insulated box is composed of aluminum silicate fiber, insulation board and iron sheet. In the evening, the electric heating device is used to provide heat energy, and the composite PCM is contained in the bottles, which stored heat energy through the heat exchange with electric heater by melting the composite PCM. The quantity of composite PCM is 43.2 kg.

Experimental procedure

The experimental equipment mainly includes temperature measuring instrument, intelligent temperature controlling switch, and automatic data acquisition system. The experimental procedures are as follows:

(a) Heat storage.

From 19:30 in the evening till 8:00 in the morning of the next day, the electric heater is switched on and controlled through the intelligent temperature controlling the switch, and the temperature in the insulated box is kept within 54 and 55 °C.

(b) Heat release.

Turn off the electric heater at 8:00 am the next morning, and turn on the switches of the blower, the air inlet temperature detector and the air outlet temperature detector to collect temperature data. The outlet temperature is recorded once every 30 s, and the inlet temperature is recorded once every 30 min.

Analysis of heat storage performance of equipment

The heat release lasts 11.5 h during the day, which releases the heat stored within the equipment by the composite

PCM, the amount of heat release Q_{sr} can be estimated by using Eq. (2),

$$Q_{sr} = m \cdot L + m \cdot (C_1 \cdot \Delta T_1 + C_s \cdot \Delta T_s) \quad (2)$$

in which, T_s is the phase transition temperature of composite PCM, here $T_s = 36$ °C; T_1 is the maximum working temperature of energy storage electric heater, 72 °C; T_2 is the minimum working temperature of energy storage electric heater, here $T_2 = 25$ °C; m is the mass quantity of composite PCM, $m = 43.2$ kg; L is the latent heat of phase the composite material, 178.02 kJ/kg (Farid et al. 2004; Jankowski and McCluskey 2013); C_1 is the specific heat capacity of the heat storage material in liquid state, 1.94 kJ/kg·°C (Farid et al. 2004; Jankowski and McCluskey 2013); C_s is the specific heat capacity of the heat storage material in solid state, $C_s = 1.69$ kJ/kg·°C (Farid et al. 2004; Jankowski and McCluskey 2013); $\Delta T_1 = T_1 - T_s$; and $\Delta T_s = T_s - T_2$.

The estimated result of Q_{sr} from Eq. (2) is 11,510.68 KJ.

Additionally, in the actual test, the average temperature of the composite PCM in the heat storage unit is 60.2 °C as the heat storage stops, and the average temperature of the material is 24.6 °C as the heat release stops.

Heat release performance of equipment

In the heat release stage, the heater is turned off, the upper vent and the lower ducted ventilating fan in the insulated box are turned on. Air enters from the upper left and exhausts from the lower right. Therefore, the composite PCM inside the heat storage unit roughly radiates heat in one direction during the heat release process, and the air is fully heated, the air temperature change at the air inlet and the air temperature at the exhaust outlet are as shown in Figures 8 and 9, respectively.

The temperature of the inlet port (Figure 8) is counted every 5 min, and the average inlet temperature is 19.4 °C. In the initial stage, since the inlet port has just been opened, there is hot air flow in the heat storage device, which causes the temperature to be unstable. After the equipment is operating stably, the air temperature at the air inlet becomes stabile gradually. The temperature of the air outlet (Figure 9) is counted every 0.5 min. When the heat is released, it takes a moment for the equipment to run to send out the heated air, so the temperature jumps to a higher value suddenly as is shown in Figure 9. It can be seen from the Figure 9 that after 11 h of operation the equipment can still be maintained at about 25 °C.

The instant increment of air heat output during daytime from insulated box can be assessed by Eq. (3),

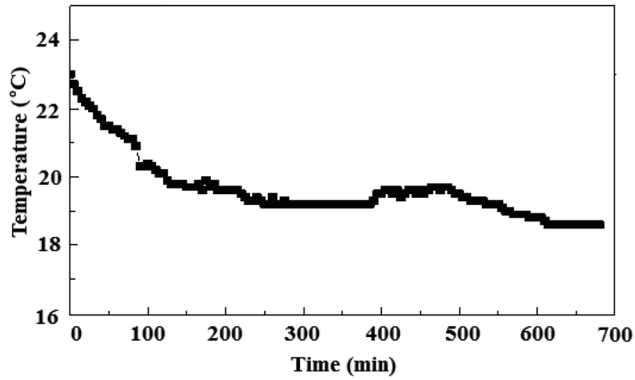


Figure 8: Temperature change of inlet air with respect to time.

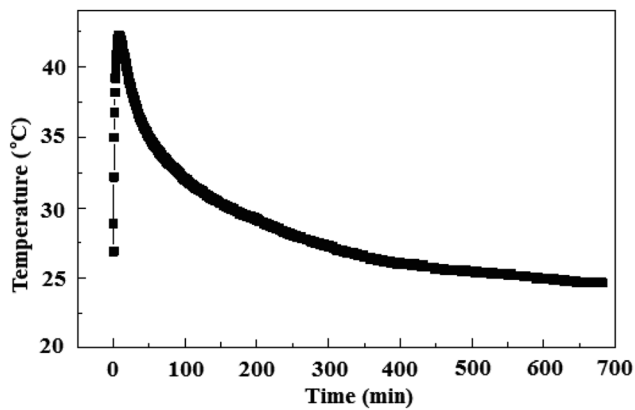


Figure 9: Temperature change of outlet air with respect to time.

$$dQ_a = V_a \cdot C_a \cdot \rho_a \cdot \Delta T \cdot dt \quad (3)$$

The integral of Eq. (3) could give the total heat release in the whole daytime,

$$Q_a = \int_T^0 V_a \cdot C_a \cdot \rho_a \cdot \Delta T \cdot dt \quad (4)$$

In Eqs. (3) and (4), Q_a is the output heat from insulated box during the daytime (11.5 h); V_a is flow rate of air of the ducted exhaust fan, here $V_a = 55 \text{ m}^3/\text{h}$; ρ_a is air density, here $\rho_a = 1.29 \text{ kg/m}^3$; T is the total time of heat release, $T = 11.5 \text{ h}$; C_a is the specific heat capacity of air, here $C_a = 1.4 \text{ kJ/kg} \cdot ^\circ\text{C}$; ΔT is the difference between the instantaneous temperature of the outlet and the instantaneous temperature of the inlet.

Finally, the estimated result of Q_a is 10,312.94 KJ.

Furthermore, the heat exchange efficiency η of the equipment can be defined as the ratio of the total released heat energy in daytime to the total restored heat energy of the composite PCM in night by Eq. (5),

$$\eta = (Q_a / Q_{sr}) \cdot 100\% \quad (5)$$

From the experimental test, the estimated value of η is 89.59%.

It can be seen that the $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ -based composite PCM and the thermal storage equipment have promising heat storage performance.

Conclusions

Thermal storage equipment with $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ -based composite PCM is designed so as to perform the heat storage and supply continuous service for succeeding application. Through the study of the heat storage behavior of the equipment, the following conclusions can be obtained,

- (1) The composite PCM composing of $\text{Na}_2\text{HPO}_4 \cdot 12\text{H}_2\text{O}$ and $\text{Na}_3\text{PO}_4 \cdot 12\text{H}_2\text{O}$ at the ratio of 7:3 has good heat storage performance with solidification temperature of 33.4°C and supercooling degree of 2.6°C .
- (2) It takes about 5.5 h to melt the composite PCM in nighttime by electric heater to store thermal energy.
- (3) The self – designed “peak load shifting” insulated box for heat storage of electricity energy has promising performance with the energy exchange efficiency of 89.59%.

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